Small-Scale CSP Systems for Industrial Process Heat on Urban Brownfields

Laura Schaefer las14@rice.edu Dept of Mechanical Engineering, Rice University

April 23, 2025





Outline

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 - Heliostats in a Combined Cycle for Waste Heat Recovery
 - Even Smaller Scale Applications: Solar Thermal-Boosted Organic Rankine Cycle
- Conclusions





Acknowledgements

- Alliance for Sustainable Energy, LLC, and the Heliostat Consortium (HelioCon) under Award A24-0362-001
- Co-PI James Elliott (Sociology



Energy Systems Lab Students

















Overall Project Scope: Introduction

Why is Industrial Process Heat So Important?



High Industrial Energy Demand

Industry accounts for ~38% of global final energy use

Industrial Process Heat (IPH)

Over 50% of U.S. manufacturing energy goes to process heat; globally IPH forms ~20% of energy demand

Urgency of Supply

Global IPH demand may grow 17% by 2030, requiring renewable heat solutions to meet these needs





Why Concentrated Solar Thermal (CST) and Solar IPH?

- Improving Energy Independence: Solar IPH reduces reliance on imported fuels.
- High-Temperature Capability: CST can supply hightemperature heat (100–1000°C) essential for diverse industrial processes.
- Renewable and Dispatchable: CST coupled with Thermal Energy Storage (TES) can deliver stable, dispatchable heat even when solar resources fluctuate.
- Economic Potential: CSP systems can achieve competitive Levelized Cost of Heat (LCOH), especially in regions with strong solar resources and rising fuel prices.
- Small-Scale Modular Systems: Compact CSP installations (≤5 MW) can be strategically deployed near industrial consumers.



Schematics of solar industrial process heat system. Source (s): Bees Group.





Why Urban Brownfields for CST Deployment?

- Underutilized Land: Urban brownfields are former industrial sites, often contaminated and unused, presenting opportunities for redevelopment into productive energy sources.
- Proximity to Industrial Demand: Brownfields are frequently located near industrial zones, aligning CST heat production directly with the end-use industrial processes.
- Lower Remediation Costs: Redeveloping brownfields involves less stringent remediation compared to residential or commercial reuse, reducing cleanup costs and duration.
- Local Community Benefits: CST deployment will help economically revive adjacent communities, creating local employment opportunities and enhancing resilience.
- Reduction of Pollution Risks: Redeveloping brownfields with clean CST technology reduces further pollution, benefiting areas with historically high exposure to industrial contaminants.





Houston brownfields. Source (s): houstontx.gov, Houston Public Media



How Does This All Come Together?

IPH Demand by Level and County

Brownfield Locations

DNI Levels









Overall Approach

- Quantify heliostat field potential for smaller-scale footprints (~10, 30, 50 acres)
- Investigate combined field-cycle installations
- Map these results to actual brownfield locations with adjacent or nearby IPH needs

| Temperature Range | Value | |
|-------------------|-------|--|
| <100°C | 33% | |
| 100–500°C | 44% | |
| 500–1000°C | 13% | |
| >1000°C | 9% | |

IPH Demand By Temperature Range







Example Site Installation



~50 acres





Many Potential Field Configurations





Material and Construction Cost Reductions



Fig. 9. Measured irradiance distribution at \sim 1700 m from the silvered glass facet (left) and the SMF1100 facet (right). The peak flux in both cases is \sim 7–8 W/m².





FIGURE 3. FE-models of investigated tower designs: a) concrete tube, diameter 22 m; b) concrete tube, diameter 15 m; c) lattice steel tower with socket; d) three-leg hybrid tower with concrete columns and steel bracings; e) cable-stayed concrete tube with two layers of cables

Tower Design https://doi.org/10.1063/1.5067098

Fig. 1. Left: heliostat 12W14 at the NSTTF in Albuquerque, NM, which was retrofitted with $3M^{TM}$ Solar Mirror Film 1100. Right: NSTTF heliostat field with location of 12W14 heliostat circled.

3M Reflective Film https://doi.org/10.1016/j.solener.2013.06.015





Combined Cycle Configurations





Fig. 1 Schematic of the investigated solar-fossil combined cycle: ISCC versus SHCC option



Combined Cycle Configurations: ORC

 Combined cycle simulation 1400 n-Heptane, ORC with IHX 1200



https://www.researchgate.net/publication/245392690 Thermodynamic Optimization of Organic Rankine Cycles at Several Condensing Temperatures Case Study_of_Waste_Heat_Recovery_in_a_Natural_Gas_Compressor_Station



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ORC Analysis







GIS Visualization

- Combining DNI map, STEP1 IPH information, national and local catalogs of brownfield locations
- Consulting with local/regional groups for input

| SHAPE | Point |
|------------|----------------------------------------|
| OBJECTID | 159435 |
| REM_PROG | Brownfields Site Assessment (BSA) |
| RN | RN100521053 |
| BSA_ID | G038 |
| SITE_NAME | HOMESTEAD PLAZA SHOPPING CENTER |
| PHYS_ADDR | 9929 HOMESTEAD RD |
| ADDR_DESC | NEAR SWC OF HOMESTEAD RD AND PARKER RD |
| CITY | HOUSTON |
| COUNTY | HARRIS |
| ZIP_CODE | 77016 |
| REGION | REGION 12 - HOUSTON |
| LAT_DD | 29.856051 |
| LONG_DD | -95.301436 |
| HORZ_METH | DOQ |
| HORZ_ACC | 5 |
| HORZ_REF | FAC_CEN |
| HORZ_DATE | 20010611 |
| HORZ_ORG | TCEQ |
| HORZ_DATUM | NAD83 |
| HORZ_DESC | Null |



Concentrating Solar-Thermal Power



Test Case Methodology: Modeling and GIS Integration

- Simulation Model: NREL's System Advisor Model to evaluate technical and economic performance.
- GIS Integration: Combined solar irradiance, IPH demand, and brownfield data to identify feasible urban deployment sites.
- Three Scenarios Modeled: 5-MW_t each
 - Parametric analysis of solar multiple (SM) and TES capacity
 - Location: Houston, TX



GIS map highlighting brownfields (black dots), IPH demand \ge 24 TBtu (by county, gray), and DNI levels

| IPH Scenario | Supply Temperature | CSP Technology | |
|--------------------|-----------------------|-------------------|--|
| Low Temperature | 150°C | PTC | |
| Medium Temperature | 300°C | PTC | |
| High Temperature | 574°C | MSPT | |





Case Study: Houston, TX



Direct Nc

Direct normal irradiance (top) and ambient temperature (bottom) in Houston, TX



Selected brownfield located in Sunnyside, Houston, TX







Parametric Analysis: SM vs TES



Parametric optimization of solar field (SM) and TES sizing. PTC-150°C (left), PTC-300°C (center), and MSPT-574°C (right).

Optimal Scenario (Minimum LCOH)

- PTC: SM is 2.6 and TES is 8
- MSPT: SM is 3.6 and TES is 14





Economic Analysis

$$LCOH = \frac{FCR \ x \ TCC + FOC}{AHP} + VOC$$

LCOH = Levelized Cost of Heat (\$/kWht) FCR = Fixed Charge Rate TCC = Total Capital Cost VOC = Variable Operating Cost (\$/kWht) FOC = Fixed Operating Cost (\$) AHP = Annual Heat Production (kWht)

| Parameter | Value |
|----------------------------|---------------------------|
| Analysis Period | 20 Years |
| Variable Operating Cost | 0.001 \$/kWh _t |
| Inflation Rate | 2.5% |
| Contingency | 7% |
| Nominal Debt Interest Rate | 7% |
| Site Improvement | 16 \$/m ² |



Results: Techno-Economic Evaluation

PTC Cases

- Both PTC plants are economically feasible and exhibit similar performance.
- Slightly better output and economics at lower temperatures.

MSPT Case

- Provides higher temperature capability (574°C), but significantly higher LCOH, making it economically less attractive.
- Requires a substantially larger capital investment and land area compared to PTC.
- Higher annual thermal output and a better capacity factor but is overshadowed by cost.

All configurations significantly reduce annual carbon emissions, contributing effectively toward industrial decarbonization goals.

Performance comparison

| Parameter | PTC 150°C | PTC 300°C | MSPT 574°C |
|----------------------------------------|--------------|--------------|---------------|
| Annual Energy (MWh _t) | 20,092.25 | 19,749.53 | 23,192.37 |
| Capacity Factor (%) | 45.9 | 45.1 | 53 |
| Thermal Storage (Hours) | 8 | 8 | 14 |
| Capital Cost (M\$) | 11.07 | 11.07 | 25.53 |
| LCOH (¢/kWh _t) | 4.39 | 4.45 | 10.30 |
| Land Required (Acres) | 16 | 16 | 35.77 |
| Annual Carbon Avoided (Metric Tons) | 11,478 | 11,282 | 13,248 |



Opportunity and Challenges

- **Economic Potential of Small-Scale:** Parabolic trough collectors (PTCs) demonstrate strong feasibility for lower-temperature industrial heat applications, highlighting an immediate opportunity for deployment.
- Cost Challenges for MSPT: MSPT technology shows high thermal performance but faces economic hurdles at smaller scales due to high LCOH.
- New Cost Models: Current economic models (e.g., SAM) require improved cost curves tailored specifically for modular, small-scale CST systems, especially for MSPT technology.
- Scaling CSP/CST Systems: Transitioning from large-scale to modular CSP/CST introduces new complexities in construction, logistics, and economics that require focused research and innovation.
- Brownfields: Accurate cost estimation incorporating land cost, site-specific remediation, and preparation costs, along with potential financial support from government programs is a challenge.





Sensitivity Analysis: MSPT Cost

Impact of Heliostat Cost

- Heliostat cost significantly influences the overall economic viability of MSPT plants.
- 17% of the total installed cost.
- SAM's current default heliostat cost is 127 \$/m²
- U.S. DOE's heliostat cost target (50 \$/m²) reduces LCOH but still results in higher costs compared to PTC configurations.

Dominance of Receiver and Tower Costs

- Receiver and tower components comprise ~48% of MSPT cost.
- Highlighting the need for new cost models tailored to smaller-scale CSP systems.

Solution

 Development of accurate cost scaling factors for heliostats, receivers, and towers at small scales.



Sensitivity analysis for heliostat field cost. Maximum is the current SAM cost, while minimum is the DOE target heliostat cost.





Conclusion and Future Work

- Small-scale CST using PTC is economically viable for low to medium temperature IPH (150°C and 300°C), with competitive LCOH.
- MSPT (574°C) system, although thermally viable, is economically challenging due to high capital cost.
- Deploying CST on urban brownfields could optimize land use, reduce pollution impacts, and economically revitalize local communities.

Moving Forward

- Development of accurate cost models specifically for modular, small-scale CSP applications.
- Detailed analysis of brownfield remediation costs and potential government incentives.
- Exploration of combined heat and power (CHP) integration to enhance system economics and performance.
- Broader geographic and technical feasibility studies to support widespread adoption.





Additional Use Cases: Example 1, Hybrid WHR System

- Primary Components
 - Gas Turbine
 - Molten Salt Power Tower
 - Thermal Storage
 - sCO₂ Brayton Cycle
- GT exhaust heats molten salt via a WHR heat exchanger.
- Mixed salt (from TES & WHR) drives the sCO₂ power block.
- Can operate in solar-only mode for flexibility.



Schematics of the proposed CSP-sCO₂ hybrid WHR system



Components

Gas Turbine Power Plants

- Gas turbines (GTs) emit high-temperature exhaust (>500°C).
- Large amount of waste heat.

Hybrid WHR

- Renewable integration enhances energy independence as well as flexibility.
- Concentrated Solar Power (CSP) suitable for GT exhaust integration.
- Higher efficiency, lower emissions, and costs.

Concentrated Solar Power

- Can reach high temperatures similar to GTs.
- Thermal energy storage (TES) Dispatchable.





Modeling Framework

A comprehensive techno-economic model.



Python-based model: GT, sCO₂, and CSP components



SolarPILOT for solar field layout optimization



PySAM for CSP components



sCO₂ model for power cycle with GT waste heat integration





Case Study (Newman, WA)

- GTs are primarily used for power in the Australian mining industry.
- Mining sites use open-cycle GTs off-grid and in remote areas.
- Rising gas prices are a major concern.
- These locations have very high solar resource for CSP deployment.
- Newman in Western Australia is one such location



Ambient temperature (top), direct normal irradiance (bottom) in Newman, WA



Results: sCO₂ Power Cycle

- Capacity = 10 MWe
- Design efficiency
 - Simple = 36.8%
 - Recompression = 42.7%
- Recompression cycle is selected due to higher efficiency.



T-S diagram for sCO₂ cycle, simple (top), recompression (bottom)





Results: Molten Salt Power Tower – CSP Component

- Solar field capacity = 30 MWt
- Thermal storage capacity = 14 hours



Thermal storage and solar field sizing (left), SolarPILOT optimized solar field layout (right)





Results: Techno-Economic Evaluation

- Output: 56,028.78 MWh/year, LCOE = \$0.0597/kWh.
- Average sCO₂ efficiency = 39%.
- CSP = largest cost share \rightarrow field/TES optimization critical.
- 29% cheaper LCOE vs CSP–Rankine (0.0843 → 0.0597 \$/kWh).



Monthly power cycle efficiency and ambient temperature



Techno-economic evaluation

| Parameter | Value |
|---------------------------------|---------------|
| Annual Production (Electricity) | 56,028.78 MWh |
| Mean Power Cycle Efficiency | 39% |
| LCOE | 0.0597 \$/kWh |
| CSP Cost | 36.78 M\$ |
| sCO ₂ Cycle Cost | 18.27 M\$ |
| Net Capital Cost | 55.74 M\$ |

Comparison with Standalone CSP-Rankine

| Parameter | CSP-sCO ₂ WHR | Standalone CSP-Rankine |
|-----------------------------|-----------------------------|---------------------------|
| Nameplate Capacity | 10 MWe | 10 MWe |
| Mean Power Cycle Efficiency | 39% | 34.24% |
| LCOE | 0.0597 \$/kWh | 0.0843 \$/kWh |
| SM | 3 | 3 |
| TES | 14 Hours | 14 Hours |
| Net Capital Cost | 55.74 M\$ | 101.76 M\$ |



Example 2, Solar Thermal-Boosted Organic Rankine Cycle for Data Centers



Rising Technology

The rise in technology has led to an exponential increase in the number of data centers.



Growing Electricity Demand

Data centers are consuming more power than some countries.



Waste Heat Dissipation

A significant amount of the consumed power is lost as waste heat due to the cooling mechanism.

Data Centers and Their Increasing Energy Appetite

Estimated electricity consumption of data centers^{*} compared to selected countries in 2022, in TWh



* Al, cryptocurrencies, traditional data centers Sources: U.S. Energy Information Administration, IEA

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Data Centers & Waste Heat Recovery (WHR)

Opportunity

- Untapped potential
- Significant energy efficiency gain
- Existing technology such as the organic Rankine cycle (ORC)

Challenges/Drawbacks

- Waste heat typically 40–60°C: hard to recover
- Low ORC supply temperatures
- Low ORC thermal efficiency
- High cost (levelized cost of electricity)

Traditional Data Center Waste Heat Recovery

Traditional ORC-based WHR system for data centers





Solar Thermal Boosted WHR

Proposed Solution

- Temperature boost using solar thermal
- High ORC thermal efficiency
- Low cost LCOE
- Increased reliability through solar integration
- Flat plate collectors (could use parabolic trough)
 - Cheapest
 - Suitable for temperatures < 100°C

Method: A techno-economic assessment



Solar thermal boosted WHR system for data centers



Methodology: Organic Rankine Cycle

- Organic Rankine Cycle
 - Python-based simulation model with hourly data
 - ORC modeled with standard thermodynamic

$$P_1, h_1, s_1 = f(x_1, T_1) \tag{1}$$

$$h_{2,s} = f(s_1, P_2) \text{ and } h_2 = h_1 + \eta_t (h_{2,s} - h_1)$$
 (2)

$$h_4 = h_3 + \frac{1}{\eta_p} (h_{4,s} - h_3) \tag{3}$$

$$\eta = \frac{W_{net}}{Q_{in}} = \frac{(h_1 - h_2) + (h_3 - h_4)}{h_1 - h_4}$$
(4)

| | Component | Cost Correlation |
|-----------------------|-------------|------------------------------------------------------------------------------------------------|
| | Condenser | $Cost_c = 12,300 \times \left(\frac{\dot{q}_c}{50}\right)^{0.76}$ |
| | Generator | $Cost_{g} = 1,850,000 	imes \left(rac{\dot{P}_{g}}{11,800} ight)^{0.94}$ |
| Economic Factors → | Pump | $log(Cost_p) = 3.3892 + 0.0536 \times log(\dot{P}_p) + 0.1538 \times \{log(\dot{P}_p)\}^2$ |
| | Turbine | $log(Cost_t) = 2.2476 + 1.4965 \times log(\dot{P}_t) - 0.1618 \times \{log(\dot{P}_t)\}^2$ |
| | Evaporator | $log(Cost_e) = 3.2138 + 0.2688 \times log(\dot{Q}_e) + 0.0796 \times \{log(\dot{Q}_{eva})\}^2$ |
| | Solar Field | $Cost_{FPC} = C_{Solar} \times A_{Collector}$ |

- Flat Plate Collectors
 - Modeled using Duffie & Beckman method
 - Useful heat gain and outlet temperature depend on:
 - Irradiance
 - Heat losses
 - Ambient conditions

$$Q_u = \dot{m}C_p \left(T_{f,out} - T_{f,in}\right) \tag{5}$$

$$\eta_c = \frac{Q_u}{A_c I_T} \tag{6}$$

$$Q_u = A_c F_R \left[S - U_L (T_{f,in} - T_a) \right]$$
(7)

| Parameter | Value |
|----------------------|-------------------------|
| Absorptivity | 0.95 |
| Transmissivity | 0.2 |
| Total mass flow rate | 0.3 (kg/s) |
| FPC area | 0.375 (m ²) |
| FPC tilt | Latitude |
| Fluid | Water |



Case Study

- Location: Ashburn, VA
- Simulation Period: One day in each season

Data Center Waste Heat Specifications

- Cooling type: Liquid cooling
 - Constant heat output
 - Waste heat supply temperature 50°C
 - Cold supply to data center 25°C



Number of data centers in each US state



Case Study (Seasonal Variation)



Solar resource in Ashburn, VA



Ambient temperature in Ashburn, VA





Results: Thermal Efficiency Enhancement

Key Findings

- Up to 8% increase in efficiency during peak solar hours
- Solar helps offset ambient temperature degradation

Solar Boosting = Better Economics

- Power output nearly doubled (189.8 → 374.5 kWh)
- 19.09% reduction in investment per kWh
- Solar adds \$20,250 but justifies cost with gains
- Solar field needs only 67.5 m² (rooftop

lool of Engineering

$$\frac{Cost Reduction (\%)}{Cost Reduction (\%)} = \left[1 - \frac{\frac{CapEx_{Solar}}{Generation_{Solar}}}{\frac{CapEx_{Non-Solar}}{Generation_{Non-Solar}}}\right] \times 100$$









Conclusions

- Multiple goals can be met:
 - Standalone power generation or heat production
 - Better utilization of empty land/space
 - Increased resiliency at economically-competitive levels
 - Boost low-temperature cycle efficiency, flexible operation
- Many questions still to be answered:
 - Largest one how do CapEx and OpEx <u>scale</u> and what are the limits of that scaling?
 - Are these viable opportunities for manufacturers and practitioners?
 - How do we increase the concentrating solar workforce to take advantage of this moment?







Thank you!

Link to Surveys Focused on Boosting the Concentrating Solar Workforce

Current College Students





RICE UNIVERSITY School of Engineering

Recent Graduates



